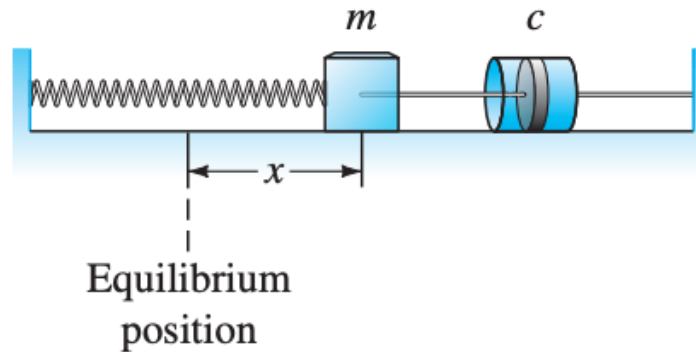


### 3.4 Mechanical Vibrations

#### Mass-spring-dashpot system



- Restorative force  $F_S = -kx$ , where  $k > 0$  is **spring constant** (Hooke's law).
- The dashpot provides force  $F_R = -cv = -c\frac{dx}{dt}$ , where  $c > 0$  is **damping constant**.
- **External force**  $F_E = F(t)$ .
- The total force acting on the mass is  $F = F_S + F_R + F_E$ .
- Using Newton's law,

$$F = ma = m\frac{d^2x}{dt^2} = mx''$$

we have the following second-order linear differential equation

$$mx'' + cx' + kx = F(t)$$

- If  $c = 0$ , we call the motion **undamped**. If  $c > 0$ , we call the motion **damped**.
- If  $F(t) = 0$ , we call the motion **free**. If  $F(t) \neq 0$ , we call the motion **forced**.

🧐 **An important note before we start analyzing the general cases:**

Rather than memorizing the various formulas given in the discussion below, it is better to practice a particular case to set up the differential equation and then solve it directly.

## 1. Free Undamped Motion ( $c = 0$ and $F(t) = 0$ )

Our general differential equation takes the simpler form

$$mx'' + kx = 0 \Rightarrow x'' + \left(\sqrt{\frac{k}{m}}\right)^2 x = 0$$

- It is convenient to define

$$\omega_0 = \sqrt{\frac{k}{m}}$$

- Then we can rewrite our equation in the form

$$x'' + \omega_0^2 x = 0$$

- Then the characteristic equation is

$$r^2 + \omega_0^2 = 0 \Rightarrow r^2 = -\omega_0^2 \Rightarrow r = \pm \omega_0 i \text{ (complex conjugate)}$$

- The general solution of this equation is

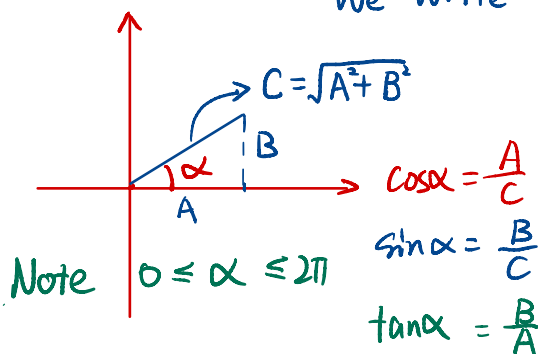
$$x(t) = A \cos \omega_0 t + B \sin \omega_0 t$$

$$\text{We write } x(t) = C \left( \frac{A}{C} \cos \omega_0 t + \frac{B}{C} \sin \omega_0 t \right)$$

$$= C (\cos \alpha \cos \omega_0 t + \sin \alpha \sin \omega_0 t)$$

$$\text{Recall } \cos(a-b) = \cos a \cos b + \sin a \sin b$$

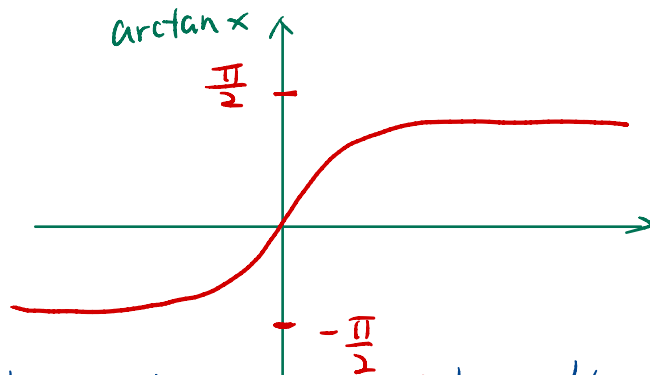
$$\text{Then } x(t) = C \cos(\omega_0 t - \alpha)$$



- Question 1. What is the values of  $C$ ?

$$C = \sqrt{A^2 + B^2}$$

- Question 2: What is the angle  $\alpha$ ?



Although  $\tan \alpha = \frac{B}{A}$ , the angle  $\alpha$  is not given by the principal branch of the inverse tangent function, which gives value only in  $(-\frac{\pi}{2}, \frac{\pi}{2})$ .

Instead,  $\alpha$  is the angle between 0 and  $2\pi$  such that  $\sin \alpha = \frac{B}{C}$ ,  $\cos \alpha = \frac{A}{C}$ , where either  $A$  or  $B$  or both

may be negative.

- Thus

$$\alpha = \begin{cases} \tan^{-1}(B/A) & \text{if } A > 0, B > 0 \text{ (first quadrant)} \\ \pi + \tan^{-1}(B/A) & \text{if } A < 0 \text{ (second or third quadrant)} \\ 2\pi + \tan^{-1}(B/A) & \text{if } A > 0, B < 0 \text{ (fourth quadrant)} \end{cases} \quad (1)$$

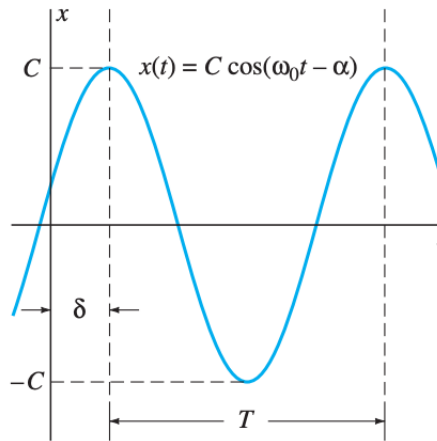
where  $\tan^{-1}(\frac{b}{a})$  is the angle in  $(-\frac{\pi}{2}, \frac{\pi}{2})$  given by a calculator or computer.

- So we have

$$x(t) = C \cos(\omega_0 t - \alpha)$$

where  $\omega$ ,  $C$  and  $\alpha$  are obtained as above.

- We call such motion **simple harmonic motion**. A typical graph of such motion is as



- To summarize, it has

Name	Symbol	Quick note
Amplitude	$C$	$C = \sqrt{A^2 + B^2}$ , where $x(t) = A \cos \omega_0 t + B \sin \omega_0 t$ is the solution for the equation $x'' + \omega_0^2 x = 0$ .
Circular frequency	$\omega_0$	$\omega_0 = \sqrt{\frac{k}{m}}$
Phase angle	$\alpha$	Obtained by formula (1) above
Period	$T = \frac{2\pi}{\omega_0}$	Time required for the system to complete one full oscillation
Frequency	$\nu = \frac{1}{T} = \frac{\omega_0}{2\pi}$ (In Hz)	It measures the number of complete cycles per second.

### Example 1

$$\cancel{m}x'' + \cancel{c}x' + \cancel{k}x = \cancel{F(t)} \quad \text{?} \quad \text{?} \quad \text{?}$$

- A body with mass  $m = 0.5$  kilogram (kg) is attached to the end of a spring that is stretched 2 meters (m) by a force of 100 newtons (N).
- It is set in motion with initial position  $x_0 = 1$  (m) and initial velocity  $v = -5$  (m/s). (Note that these initial conditions indicate that the body is displaced to the right and is moving to the left at time  $t = 0$ .)
- Find the position function of the body in the form  $C \cos(\omega_0 t - \alpha)$  as well as the amplitude, frequency and period of its motion.

ANS: Since  $F_s = 100 = k \cdot 2 \Rightarrow k = 50 \text{ N/m}$

Then we have

$$0.5x'' + 50x = 0$$

$$\Rightarrow x'' + 100x = 0 \quad (x'' + \omega_0^2 x = 0)$$

From our previous discussion, we have

$$\omega_0^2 = 100 \Rightarrow \omega_0 = 10 \text{ rad/s.}$$

Thus it will oscillate with period  $T = \frac{2\pi}{\omega_0} = \frac{\pi}{5}$ .

with frequency  $\nu = \frac{1}{T} = \frac{5}{\pi} \approx 1.5915 \text{ Hz.}$

The char. eq  $r^2 + 100 = 0 \Rightarrow r = \pm 10i$ .

Thus we have  $x(t) = A \cos 10t + B \sin 10t$ . where A and B are constants.

As  $x(0) = 1$ ,  $A = 1$ .  $x'(t) = -10A \sin 10t + 10B \cos 10t$

$x'(0) = 10B = -5$ ,  $\Rightarrow B = -\frac{1}{2}$ .

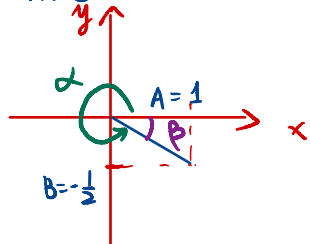
Then  $x(t) = \cos 10t - \frac{1}{2} \sin 10t$

The amplitude of the motion is  $C = \sqrt{A^2 + B^2} = \sqrt{1 + \frac{1}{4}} = \frac{\sqrt{5}}{2} \text{ m.}$

Thus we have

$$x(t) = \frac{1}{2}\sqrt{5} \left( \frac{2}{\sqrt{5}} \cos 10t - \frac{1}{\sqrt{5}} \sin 10t \right)$$

cos α      sin α



By the graph, we have  
assume  $0 \leq \beta \leq \frac{\pi}{2}$

$$\alpha = 2\pi - \beta = 2\pi - \tan^{-1} \frac{\frac{1}{2}}{1} = 2\pi - \tan^{-1} \frac{1}{2} \approx 5.8195 \text{ rad.}$$



$$\alpha = \begin{cases} \tan^{-1}(B/A) & \text{if } A > 0, B > 0 \text{ (first quadrant)} \\ \pi + \tan^{-1}(B/A) & \text{if } A < 0 \text{ (second or third quadrant)} \\ \underline{2\pi + \tan^{-1}(B/A)} & \text{if } A > 0, B < 0 \text{ (fourth quadrant)} \end{cases}$$

We can also use formula 1. with  $A=1>0$ ,  $B=-\frac{1}{2}<0$

$$\begin{aligned} \alpha &= 2\pi + \tan^{-1} \frac{B}{A} \\ &= 2\pi + \tan^{-1} \frac{-\frac{1}{2}}{1} \\ &= 2\pi + \tan^{-1}(-\frac{1}{2}) \\ &\approx 5.8195 \text{ rad} \end{aligned}$$

Thus  $x(t) = \frac{\sqrt{5}}{2} \cos(10t - 5.8195)$

## 2. Free Damped Motion ( $c > 0$ and $F(t) = 0$ )

In this case, we consider

$$mx'' + cx' + kx = 0$$

Let  $\omega_0 = \sqrt{k/m}$  and  $p = \frac{c}{2m} > 0$ . We have

$$x'' + 2px' + \omega_0^2 x = 0$$

The characteristic equation

$$r^2 + 2pr + \omega_0^2 = 0$$

has roots

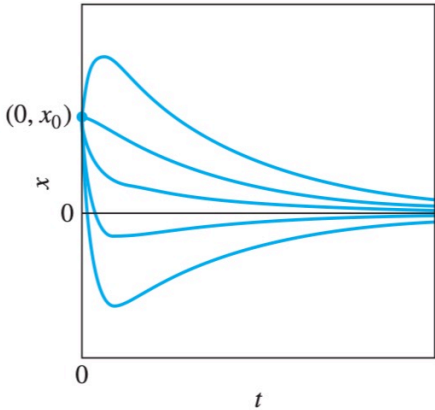
$$r_1, r_2 = -p \pm \sqrt{p^2 - \omega_0^2} \quad (2)$$

Note

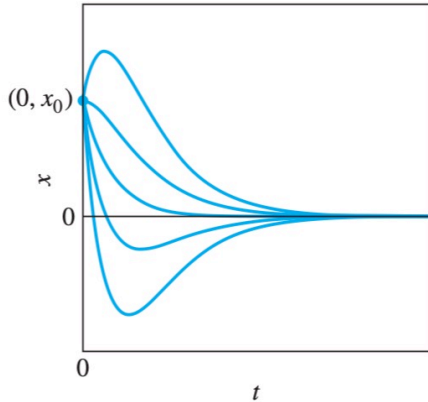
$$p^2 - \omega_0^2 = \frac{c^2 - 4km}{4m^2}$$

We have the following three cases.

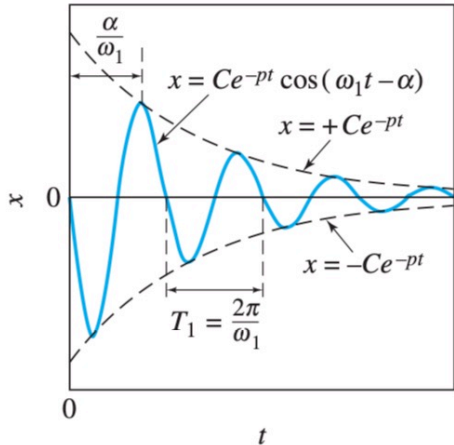
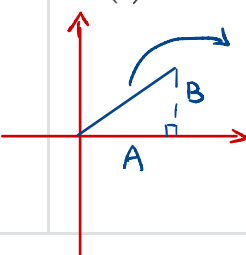
### Case 1. Overdamped ( $c^2 > 4km$ , two distinct real roots)

Figure	Analysis
 <p><b>FIGURE 3.4.7.</b> Overdamped motion: <math>x(t) = c_1 e^{r_1 t} + c_2 e^{r_2 t}</math> with <math>r_1 &lt; 0</math> and <math>r_2 &lt; 0</math>. Solution curves are graphed with the same initial position <math>x_0</math> and different initial velocities.</p>	<p>Eq(3) gives two distinct real roots <math>r_1</math> and <math>r_2</math> (both <math>&lt; 0</math>). The position function</p> $x(t) = c_1 e^{r_1 t} + c_2 e^{r_2 t}$ <p>Note</p> $\lim_{t \rightarrow \infty} x(t) = 0$ <p>(The object will go to the equilibrium position without any oscillations.)</p>

### Case 2. Critically damped ( $c^2 = 4km$ , repeated real roots)

Figure	Analysis
 <p><b>FIGURE 3.4.8.</b> Critically damped motion: <math>x(t) = (c_1 + c_2 t)e^{-pt}</math> with <math>p &gt; 0</math>. Solution curves are graphed with the same initial position <math>x_0</math> and different initial velocities.</p>	<p>Eq(3) has roots <math>r_1 = r_2 = -p</math>. The general solution for the position function.</p> $x(t) = e^{-pt}(c_1 + c_2 t)$ <p>and</p> $\lim_{t \rightarrow \infty} x(t) = 0$ <p>The resistance of the dashpot is just enough to damp out any oscillations.</p>

### Case 3. Underdamped ( $c^2 < 4km$ , two complex roots)

Figure	Analysis
 <p><b>FIGURE 3.4.9.</b> Underdamped oscillations:  <math>x(t) = Ce^{-pt} \cos(\omega_1 t - \alpha)</math>.</p>	<p>Eq(3) has roots <math>r_{1,2} = -p \pm i\sqrt{\omega_0^2 - p^2} = -p \pm \omega_1 t</math>, where <math>\omega_1 = \sqrt{\omega_0^2 - p^2}</math></p> <p>The general solution for the position function</p> $x(t) = e^{-pt}(A \cos \omega_1 t + B \sin \omega_1 t)$ <div style="text-align: center;">  <math display="block">C = \sqrt{A^2 + B^2}</math> </div> $x(t) = e^{-pt} C \left( \frac{A}{C} \cos \omega_1 t + \frac{B}{C} \sin \omega_1 t \right)$ $= C e^{-pt} (\cos \alpha \cos \omega_1 t + \sin \alpha \sin \omega_1 t)$ $\Rightarrow x(t) = C e^{-pt} \cos(\omega_1 t - \alpha)$

**Example 2** Suppose that the mass in a mass-spring-dashpot system with  $m = 6$ ,  $c = 7$ , and  $k = 2$  is set in motion with  $x(0) = 0$  and  $x'(0) = 2$ .

(a) Find the position function  $x(t)$ .

(b) Find how far the mass moves to the right before starting back toward the origin.

Ans: We have  $6x'' + 7x' + 2x = 0$ ,  $x(0) = 0$ ,  $x'(0) = 2$ .

The char. eqn  $6r^2 + 7r + 2 = 0$ .

$$\Rightarrow r_{1,2} = \frac{-7 \pm \sqrt{49 - 48}}{12} = \frac{-7 \pm 1}{12} = -\frac{2}{3}, -\frac{1}{2}$$

Thus  $x(t) = C_1 e^{-\frac{2}{3}t} + C_2 e^{-\frac{1}{2}t}$

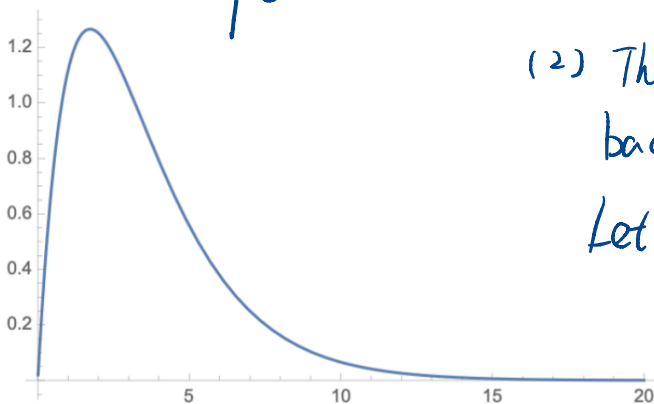
$$x(0) = 0, \quad C_1 + C_2 = 0$$

$$\text{Since } x'(0) = 2, \quad x'(t) = -\frac{2}{3}C_1 e^{-\frac{2}{3}t} - \frac{1}{2}C_2 e^{-\frac{1}{2}t}$$

$$x'(0) = -\frac{2}{3}C_1 - \frac{1}{2}C_2 = 2$$

$$\Rightarrow \begin{cases} C_1 = -12 \\ C_2 = 12 \end{cases}$$

$$\text{Thus } x(t) = -12e^{-\frac{2}{3}t} + 12e^{-\frac{1}{2}t}$$



(2) The mass starts to move back when  $x'(t) = 0$ .

$$\text{Let } x'(t) = 8e^{-\frac{2}{3}t} - 6e^{-\frac{1}{2}t} = 0$$

$$\Rightarrow t = 6 \ln \frac{4}{3} \approx 1.72609 \text{ s.}$$

Thus we have

$$x(6 \ln \frac{4}{3}) = \frac{81}{64} \approx 1.26563 \text{ m}$$

**Example 3** In the following problems (a) and (b), a mass  $m$  is attached to both a spring (with given spring constant  $k$ ) and a dash-pot (with given damping constant  $c$ ). The mass is set in motion with initial position  $x_0$  and initial velocity  $v_0$ .

- (1) Find the position function  $x(t)$  and determine whether the motion is *overdamped*, *critically damped*, or *underdamped*. If it is underdamped, write the position function in the form  $x(t) = C_1 e^{-\gamma t} \cos(\omega_1 t - \alpha_1)$ .
- (2) Find the undamped position function  $u(t) = C_0 \cos(\omega_0 t - \alpha_0)$  that would result if the mass on the spring were set in motion with the same initial position and velocity, but with the dashpot disconnected (so  $c = 0$ ).
- (3) Construct a figure that illustrates the effect of damping by comparing the graphs of  $x(t)$  and  $u(t)$ .

(a)  $m = 1, c = 4, k = 3; x_0 = 2, v_0 = 0$

Ans: (i) With damping.

$$x'' + 4x' + 3x = 0, \quad x(0) = 2, \quad x'(0) = 0.$$

$$\Rightarrow r^2 + 4r + 3 = (r+1)(r+3) = 0$$

$$\Rightarrow r_1 = -1, \quad r_2 = -3.$$

$$\text{Thus } x(t) = C_1 e^{-t} + C_2 e^{-3t}$$

$$\text{As } x(0) = 2, \quad \underline{C_1 + C_2 = 2}$$

$$\text{As } x'(0) = 0, \quad x'(t) = -C_1 e^{-t} - 3C_2 e^{-3t}$$

$$\underline{x'(0) = -C_1 - 3C_2 = 0}$$

$$\Rightarrow \begin{cases} C_1 + C_2 = 2 \\ -C_1 - 3C_2 = 0 \end{cases} \Rightarrow \begin{cases} C_1 = 3 \\ C_2 = -1 \end{cases}$$

Since we have  
two real solutions  
 $\downarrow r_1 \neq r_2$

Thus  $x(t) = 3e^{-t} - e^{-3t}$ , which describes **overdamped motion**

(2) Without damping ( $C=0$ )

We have  $x'' + 3x = 0$ .

The char. eqn is  $r^2 + 3 = 0 \Rightarrow r = \pm\sqrt{3}i$ .

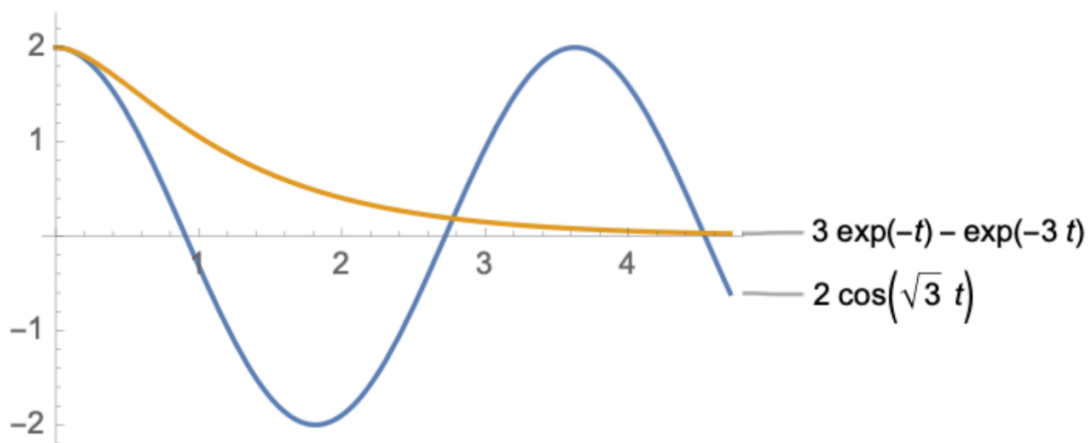
Thus  $x(t) = A \cos \sqrt{3}t + B \sin \sqrt{3}t$ .

$$x'(t) = -\sqrt{3} A \sin \sqrt{3}t + \sqrt{3} B \cos \sqrt{3}t.$$

$$x(0) = 2, \quad A = 2$$

$$x'(0) = 0, \quad \sqrt{3} B = 0, \quad \Rightarrow B = 0$$

Thus  $x(t) = 2 \cos \sqrt{3}t \rightarrow$  Note this solution is already in the form  $C_0 \cos(\omega_0 t - \alpha_0)$



(b)  $m = 1, c = 2, k = 10; x_0 = 2, v_0 = 4$

Ans: (1) With damping  $c = 2$ .

We have

$$x'' + 2x' + 10x = 0, \quad x(0) = 2, \quad x'(0) = 4.$$

The char. eqn is

$$r^2 + 2r + 10 = 0$$

$$\Rightarrow r_{1,2} = \frac{-2 \pm \sqrt{4 - 40}}{2} = -1 \pm 3i.$$

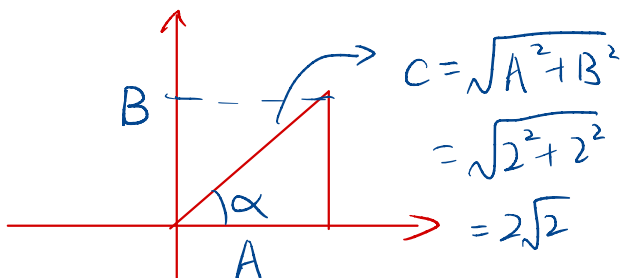
Thus  $x(t) = e^{-t} (A \cos 3t + B \sin 3t)$

As  $x(0) = 2$ ,  $A = 2$ .

$$x'(t) = -e^{-t} (A \cos 3t + B \sin 3t) + e^{-t} (-3A \sin 3t + 3B \cos 3t)$$

As  $x'(0) = 4$ ,  $-A + 3B = 4 \Rightarrow B = 2$ .

So  $x(t) = e^{-t} (2 \overset{A}{\cos 3t} + 2 \overset{B}{\sin 3t})$  underdamped  
since the char. eqn has complex conjugates solutions.



$$\tan \alpha = \frac{2}{2} = 1$$

Notice  $\alpha$  is in the 1st quadrant,  $\alpha = \frac{\pi}{4}$

$$x(t) = 2\sqrt{2} e^{-t} \cos \left( 3t - \frac{\pi}{4} \right)$$

(2) Without damping ( $c = 0$ ).

We have

$$x'' + 10x = 0, \quad x(0) = 2, \quad x'(0) = 4.$$

$$\Rightarrow r^2 + 10 = 0 \Rightarrow r = \pm \sqrt{10} i.$$

$$\text{Then } x(t) = A \cos \sqrt{10} t + B \sin \sqrt{10} t.$$

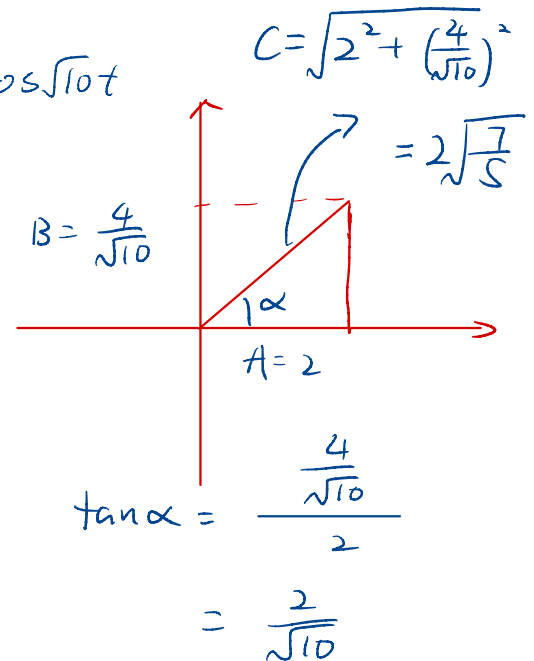
$$\text{As } x(0) = 2, \quad A = 2$$

$$\text{As } x'(0) = 4, \quad x'(t) = -\sqrt{10} A \sin \sqrt{10} t + \sqrt{10} B \cos \sqrt{10} t$$

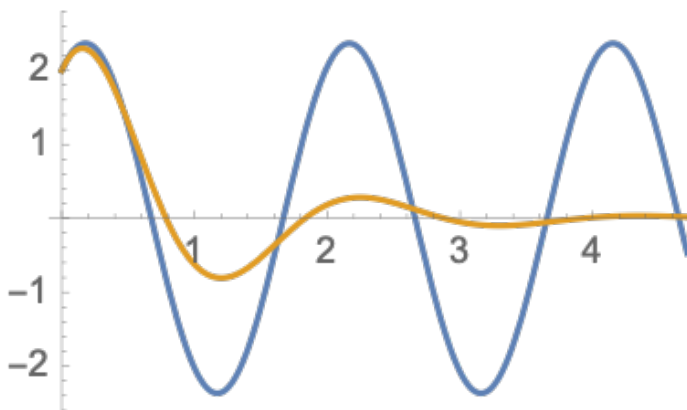
$$\text{Thus } \sqrt{10} B = 4 \Rightarrow B = \frac{4}{\sqrt{10}}$$

$$\text{So } x(t) = \underset{\substack{\uparrow \\ A}}{2} \cos \sqrt{10} t + \underset{\substack{\uparrow \\ B}}{\frac{4}{\sqrt{10}}} \sin \sqrt{10} t$$

$$\text{Thus } x(t) = 2\sqrt{\frac{7}{5}} \cos(\sqrt{10} t - 0.569)$$



$$\alpha \approx 0.5639$$



$$\begin{aligned} & \exp(-t) (2 \cos(3t) + 2 \sin(3t)) \\ & 2 \cos(\sqrt{10} t) + \frac{4 \sin(\sqrt{10} t)}{\sqrt{10}} \end{aligned}$$